

General approach - compare installed braking performance with vehicle characteristic.

Brake Analysis.

$$\theta_1 = 34^\circ \quad \theta_2 = 120^\circ \quad \mu = 0.4$$

$$I_{gg} = 1.3660 \quad I_{ss} = 1.2184 \quad I_{se} = 0.25$$

$$\therefore m = \frac{1}{0.4} \times \frac{125}{125} \times 1.2184 / 1.3660 = 1.6055$$

$$n = 1 - \left(\frac{125}{125} \right) \left(\frac{0.25}{1.3660} \right) = 0.8682$$

Both shafts (1) & (2) of any of the brakes are subjected to the same activation moment of $M = (\rho A)_{hyd} * e$, $e = 160\text{mm}$.

So from (13) with $\tau_1 = \tau_2$:

$$T_0 = (\rho A)_{hyd} e \left[\frac{1}{m-n} + \frac{1}{mn} \right] = \frac{2me}{m^2-n^2} (\rho A)_{hyd}. \quad \text{Eq (2)}$$

Letting R be the radius of the shafts, the braking force will be, for two wheels:-

$$F = 2 \cdot T_0 / R = [4me / (m^2-n^2)] (\rho A)_{hyd} / R.$$

Brake Characteristic

Since the same pressure is applied to both rear & front sets, (until maxima are reached):

$$\frac{F_R}{F_F} = \frac{F_R}{F_F} = \frac{(A_R/A_F)}{\rho F} - \text{from above} = 0.5$$

i.e. braking is proportional, with limits from (I) corresponding to maximum hydraulic press:

$$\hat{P}_R = \hat{F}_R / W = [4me / (m^2-n^2)] (\hat{\rho} A)_{hyd} / WR.$$

$$= \frac{4 \times 1.6055 \times 0.16}{1.6055^2 - 0.8682} * \frac{4 \times \frac{1}{3} \times 20.5^2}{1.2 \times 10^3 \times 9.81 \times 0.32} = 0.1975$$

$$\hat{P}_F = \frac{---}{---} * \frac{55 - 2.9^2}{---} = 0.5433$$

- a) These are superimposed on the vehicle characteristic ($c_F = c_R = 1/2$, $k = 1/4$) below. Since the braking characteristic lies everywhere below the vehicle characteristic, the brakes are safe.

- b) The max. deceleration occurs at the point A.

$$\hat{z} = 0.1975 + 0.5433 \text{ (eq (23))} = 0.740$$

ie max. deceleration = $0.74 \times 0.81 = 7.3 \text{ m/s}^2$

- c) From (7b), (5b) & (9)

$$M = (\rho A)_{hyd} * e$$

$$= \mu N r I_S (m - \delta d n)$$

where

$$N = \rho_0 W r = \rho_m W r \frac{\theta_2 - \theta_1}{\pi^2}$$

Whence max ρ_m , in self-actuating case:

$$\hat{\rho}_m = \frac{2(\rho A)_{hyd}}{\mu W r^2 (\theta_2 - \theta_1) Y_{mn}}$$

Showing for front & rear:

$$\hat{\rho}_{mF} = 0.73 \quad \hat{\rho}_{mR} = 1.34 \text{ MPa}$$

