

1. $2cz = r^2 \therefore cz' = r ; cz'' = 1 ; ' = d/dr$

Equation of normal through (r, z)

$z' = r / (z_0 - z)$ i.e. $z_0 = z + r/z'$

Length of normal, r_θ , is given by

$r_\theta^2 = (z_0 - z)^2 + r^2 = r^2 [1 + 1/(z')^2] = r^2 + c^2$

& $r_\theta = (r^2 + c^2)^{1/2}$

The radius of curvature, r_ρ , is

$r_\rho = [1 + (z')^2]^{3/2} / z''$ and from above:
 $= [1 + (r/c)^2]^{3/2} c = (r^2 + c^2)^{3/2} / c^2$

Substitute into membrane stress equations (2):-

$\sigma_\rho = (p/2t) r_\theta = k (r^2 + c^2)^{1/2} ; k = p/2t$

$\sigma_\theta = \sigma_\rho [2 - r_\theta/r_\rho] = k (r^2 + c^2)^{1/2} [2 - c^2/(r^2 + c^2)]$
 $= k (2r^2 + c^2) / (r^2 + c^2)^{1/2}$ QED.

For the design of the glass dome, fairly uniformly the membrane stresses are maximum when r is maximum.

Inserting values, assuming pressure and stresses in MPa and all dimensions in mm

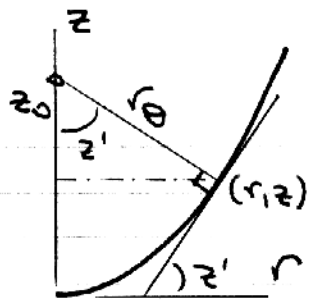
$\sigma_\rho = k (150^2 + 60^2)^{1/2} = 161.6 k$

$\sigma_\theta = k (2 \times 150^2 + 60^2) / 161.6 = 300.8 k$ } so the hoop stress σ_θ is greater

Setting $\sigma_\theta = 35 \text{ MPa}$, the design stress

$p = \rho g h = 10 (\text{kN/m}^3) \times 200 (\text{m}) = 2 \text{ MPa}$

then $35 = 300.8 p/2t ; t = 300.8 \times 2 / 2 \times 35 = 8.6 \text{ mm}$



2(a) σ_ρ may be found from free body sketched, then σ_θ follows from membrane equation.

For the free body, $p = \rho (h - z)$

$\therefore \sigma_\rho \cos \alpha \cdot t \cdot 2\pi r = pA + W = pA + \rho \frac{1}{3} A z$ (vol of cone)
 where $A = \pi r^2$ and $r = z \tan \alpha$.

$\therefore 2\pi r t \cos \alpha \cdot \sigma_\rho = A [\rho (h - z) + \frac{1}{3} \rho z] = \pi r^2 \rho (3h - 2z) / 6$
 i.e. $\sigma_\rho = k (3h - 2z) z ; k = \rho \tan \alpha / 6 t \cos \alpha$

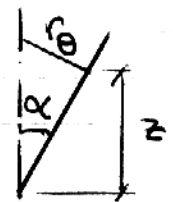
Now require the radii r_θ, r_ρ for membrane equation

The radius of curvature of the meridian $\rightarrow \infty$

i.e. $r_\rho \rightarrow \infty$. From the sketch, $r_\theta = z \tan \alpha \sec \alpha$

So the membrane equation (ii) becomes:-

$\sigma_\theta = r_\theta [p/t - \sigma_\rho / r_\rho] = (p/t) r_\theta ; p = \rho (h - z)$
 $= \rho (h - z) t \cdot z \sin \alpha / \cos^2 \alpha = k 6z(h - z)$



(b)

Putting the membrane equations & $\sigma_z = 0$ into dist² equation

$2\sigma_E^2 = (\sigma_\theta - 0)^2 + (\sigma_\rho - 0)^2 + (\sigma_\theta - \sigma_\rho)^2$

$2(\sigma_E/kz)^2 = 36(h - z)^2 + (3h - 2z)^2 + [6h - 6z - 3h + 2z]^2$

$\sigma_E^2 = (kz)^2 [27h^2 - 54hz + 28z^2]$

Differentiating for max σ_E :-

$d(\sigma_E^2)/dz = 0 = 54z^2 - 81hz + 27h^2 ; h = 6.5 \text{ m}$

$\therefore z = 6.015 \text{ m or } 3.386 \text{ m}$

2 (concl'd) It may easily be shown that the latter smaller value of z corresponds to max σ_E , i.e. σ^* . So

$$\sigma^*/k = 3.386 [27 \times 6.5^2 - 54 \times 6.5 \times 3.386 + 28 \times 3.386^2]^{1/2} = 56.0 \text{ m}^2$$

$$\Delta = \sigma^* b t \cos \alpha / \rho \tan \alpha \quad ; \quad \rho = 9.81 \text{ kN/m}^3$$

$$\therefore t = \frac{56.0 \times 9.81 \times 10^3 \times 1}{6 \times 65 \times 10^6 \times 1/\sqrt{2}} = 1.99 \text{ mm}$$

At water level, $z = h$, so membrane stresses from above

$$\sigma_\theta = 0 \quad ; \quad \sigma_\phi = k h^2$$

$$\therefore \epsilon_\phi = \epsilon_\theta = (\sigma_\theta - \nu \sigma_\phi) / E = -\nu \rho h^2 \tan \alpha / 6 E t \cos \alpha$$

$$= -0.3 \times \frac{9.81 \times 10^3}{\text{m}^3} \times \frac{6.5^2}{\text{m}^2} \times \frac{1}{6} \times \frac{207 \times 10^9}{\text{N}^2} \times \frac{2 \times 10^{-3}}{\text{m}} \times \frac{1}{\sqrt{2}}$$

$$= -71 \times 10^{-6} \quad \text{i.e. compression of } 71 \text{ } \mu\text{strain.}$$