

4. Assume $0.75F_p \leq F_i \leq 0.9F_p$ where here

$$F_p = A_s S_p = 245 \times 590 = 145 \text{ kN.}$$

$$\therefore 109 \leq F_i \leq 130 \text{ kN say } F_i \approx 120 \text{ kN}$$

IF friction is "normal", then, from (2)

$$T = K F_i d = 0.2 \times 120 \times 20 = 480 \text{ Nm}$$

Use a tightening torque of 480 Nm, but realise the large errors in $F_i = 120 \text{ kN}$ which can arise in practice.

In order to find out how the load is shared between bolts and joint members we have to examine their stiffnesses. The joint is metal to metal so we may use (A) for frustum length $48/2 = 24 \text{ mm}$

$$k_{j1 \text{ or } 2} = 207 \times 20 \times \frac{0.702 + 0.697 \frac{24}{24}}{1 - 0.12 \times \frac{24}{24}} = 5740 \text{ kN/mm}$$

so, for the two joint cores in series

$$1/k_j = 1/k_{j1} + 1/k_{j2} \Rightarrow k_j = \underline{2870 \text{ kN/mm}}$$

For the bolt, half the load & nut length is approx $0.5 \times 0.9 \times 20 = 9 \text{ mm}$.

Assume 2 exposed threads, of length $2 \times 2.5 = 5 \text{ mm}$

shank $L_s = 48 - 5 + 9 = 52 \text{ mm}$ $A = \frac{\pi}{4} \times 20^2 = 314 \text{ mm}^2$

threads $L_t = 5 + 9 = 14 \text{ mm}$ $A = A_s = 245 \text{ mm}^2$

$$1/k_b = 1/k_{\text{shank}} + 1/k_{\text{threads}} = \sum L/AE$$

$$= (52/314 + 14/245) / 207$$

$$\Rightarrow k_b = \underline{929 \text{ kN/mm}}$$

so, for one of the two joint assemblies

$$1/k_e = 1/k_b + 1/k_j = 1/929 + 1/2870 \Rightarrow k_e = 702 \text{ kN/mm}$$

From (3) with $P = 20 \text{ kN}$ (symmetry assumed)

$$F_b = F_i + P \frac{k_e}{k_j} = 120 + 20 \times \frac{702}{2870} = 125 \text{ kN}$$

$$F_j = F_i - P \frac{k_e}{k_b} = 120 - 20 \times \frac{702}{929} = 105 \text{ kN}$$

Note that the nicety of considering the stiffness of exposed threads, and of $1/2$ bolt head & nut, is probably unnecessary.